Section Contents

Speed Reducer Nomenclature

AXIAL MOVEMENT – Endwise movement of input or output shafts, sometimes called endplay, is usually expressed in thousands of an inch.

EFFICIENCY – The amount of output power of the reducer as compared to the amount of input power. It is usually stated as a percentage.

Example:

Input HP = 1

Output HP = .75

(75/100) x (100) = 75% Efficiency

BACKLASH – Rotational movement of the output shaft when holding the input shaft stationary and rotating the output shaft alternately clockwise and counter clockwise. Backlash may be expressed in thousands of an inch measured at a specific radius at the output shaft.

CENTER DISTANCE – On a single reduction reducer, this is the distance between the center lines of the input and output shafts. Shaft center lines may be parallel or at right angles to one another. The center distance of multiple stage reducers usually refers to the lowest speed stage (last reduction).

THRUST LOAD – Forces imposed on a shaft parallel to the shaft axis. Such a force is called a thrust load. It is often encountered on shafts driving mixers, fans, blowers and similar machines. When a thrust load acts on a speed reducer, you must be sure that the thrust load rating of the reducer is high enough that it's shafts and bearings can absorb the load.

MECHANICAL RATING – The maximum power or torque that a speed reducer can transmit, based on the strength and durability of its components, is it's mechanical rating. Obviously, the reducer may be rated no higher than the strength or durability of its weakest component. Reducers typically have a safety margin of two to three on their mechanical ratings. Thus, a reducer can withstand momentary overloads of 200-300% of its mechanical rating during a startup or other brief overload situations.

THERMAL RATING – The maximum power or torque that a speed reducer can transmit continuously, based on its ability to dissipate heat generated by friction, is called its thermal rating.

PRIME MOVER – The machine that provides power to a drive is its prime mover. The most frequently encountered prime movers include electric motors, internal combustion engines, hydraulic motors and air motors. The type of prime mover used can affect the speed reducer during operation. For example, an electric motor runs relatively smoothly in comparison to an internal combustion engine.

MOUNTING POSITION – The relationship of the input and output shafts relative to the floor line.

INPUT HORSEPOWER – The amount of power applied to the input shaft of a reducer by the prime mover is its input horsepower. It is often used as a selection basis for power transmission components, and it appears in the rating tables of drive manufacturer's published data. Remember that input horsepower ratings represent the maximum amount of power that the reducer can safely handle.

OUTPUT HORSEPOWER – The amount of power available at the output shaft of a reducer is its output horsepower. Due to losses caused by inefficiency, output horsepower is always less than input horsepower.

OVERHUNG LOAD – The input or the output shaft of a speed reducer can be subject to an overhung load; that is, to a force applied at right angles to the shaft, beyond its outermost bearing. Such a force is a shaft bending load resulting from a gear, pulley, sprocket or other external drive member. Besides the tendency to bend the shaft, the overhung load (that is, the radial force on the shaft) is reacted to by the shaft in it's bearings. Therefore, the overhung load creates loads that the bearings must be able to support without damage.

SERVICE FACTORS – Numbers which modify the loads which must be considered in selecting a speed reducer are called service factors. They vary with the type of service in which the reducer is to be used, the kind of prime mover involved and the duty cycle. The service factor can be a multiplier applied to the known load, which redefines the load in accordance with the conditions at which the drive will be used, or it can be a divisor applied to catalog reducer ratings, thus redefining the rating in accordance with drive conditions. The service factor is usually applied to the speed reducer, but can also be applied to the name plate rating of the prime mover.

REDUCTOR® – Boston Gear's registered trademark for a speed reducer having a projecting input shaft suitable for mounting a coupling, sprocket, pulley or gear.

FLANGED REDUCTOR – Boston Gear's name for a reductor furnished with an input flange suitable for attaching a face mounted motor.

RATIOMOTOR®– Boston Gear's registered trademark for a motorized reducer consisting of a flanged reductor and face mounted motor assembled, sometimes referred to as a gearmotor.

SELF-LOCKING ABILITY – Boston 700 Series reducers, under no conditions should be considered to hold a load when at rest.

BACK-DRIVING – This is the converse of self-locking. Depending upon ratio and many variables, it is difficult to predict the back-driving capability of a 700 Series reducer. Worm gear reducers are not intended to be used as speed increasers. Consult factory for back-driving applications.

POWER is the rate of doing work.

WORK is the exerting of a FORCE through a DISTANCE. ONE FOOT POUND is a unit of WORK. It is the WORK done in exerting a FORCE OF ONE POUND through a DISTANCE of ONE FOOT.

THE AMOUNT OF WORK done (Foot Pounds) is the FORCE (Pounds) exerted multiplied by the DISTANCE (Feet) through which the FORCE acts.

THE AMOUNT OF POWER used (Foot Pounds per Minute) is the WORK (Foot Pounds) done divided by the TIME (Minutes) required.

$$\text{POWER (Foot Pounds per Minute)} = \frac{\text{WORK (Ft. Lbs.)}}{\text{TIME (Minutes)}}$$

POWER is usually expressed in terms of HORSEPOWER.

HORSEPOWER is POWER (Foot Pounds per Minute) divided by 33,000.

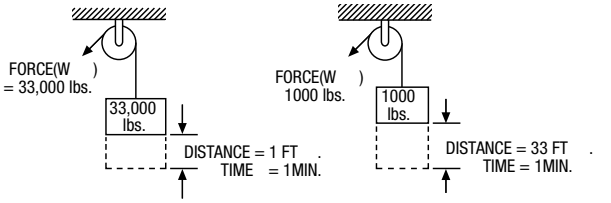
HORSEPOWER (HP)

$$= \frac{\text{POWER (Ft. Lbs. per Minute)}}{33{,}000}$$

$$= \frac{\text{WORK (Ft. Pounds)}}{33{,}000 \times \text{TIME (Min.)}}$$

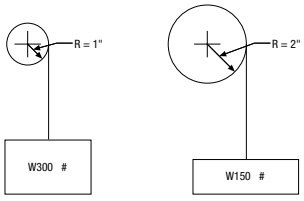
$$= \frac{\text{FORCE (Lbs.)} \times \text{DISTANCE (Feet)}}{33{,}000 \times \text{TIME (Min.)}}$$

Illustration of horsepower



$$HP = \frac{33{,}000 \times 1}{33{,}000 \times 1} = 1 \text{ HP} \qquad HP = \frac{1000 \times 33}{33{,}000 \times 1} = 1 \text{ HP}$$

TORQUE (T) is the product of a FORCE (W) in pounds, times a RADIUS (R) in inches from the center of shaft (Lever Arm) and is expressed in Inch Pounds.



T = WR
= 300 x 1 = 300 In. Lbs.

T = WR
= 150 x 2 = 300 In. Lbs.

If the shaft is revolved, the FORCE (W) is moved through a distance, and WORK is done.

$$\text{WORK (Ft. Lbs.)} = W \times \frac{2\pi R}{12} \times \text{No. of Rev. of shaft}$$

When WORK is done in a specified TIME, POWER is used.

$$\text{POWER (Ft. Pounds per Minute)} = W \times \frac{2\pi R}{12} \times \text{RPM}$$

Since (1) HORSEPOWER = 33,000 Ft. Pounds per Minute

$$\text{Horsepower (HP)} = W \times \frac{2\pi R}{12} \times \frac{\text{RPM}}{33{,}000} = \frac{W \times R \times \text{RPM}}{63{,}025}$$

but TORQUE (Inch Pounds) = FORCE (W) x RADIUS (R)

$$\text{Therefore HORSEPOWER (HP)} = \frac{\text{TORQUE (T)} \times \text{RPM}}{63{,}025}$$

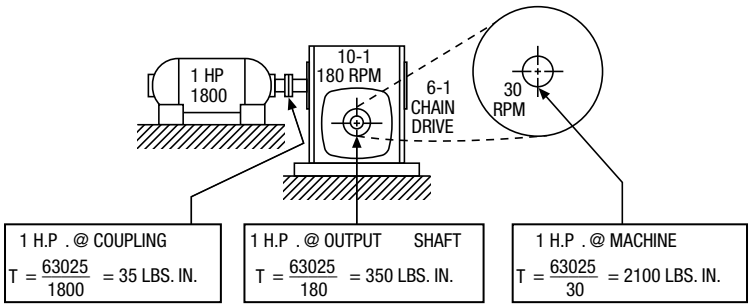
Where total reductions are small, 50 to 1 or less, HP figures are commonly used. Higher reductions require that TORQUE figures be used to select drive components, because with large reductions, a small motor can produce extremely high TORQUE at the final low speed. For example, 1/12 HP reduced to 1 RPM using the formula below and neglecting friction:

$$HP = \frac{\text{TORQUE} \times \text{RPM}}{63{,}025} \text{ or } \text{TORQUE} = \frac{63{,}025 \times HP}{\text{RPM}}$$

$$\text{TORQUE} = \frac{63{,}025 \times 1/12}{1} = 5{,}252 \text{ In. Lbs.}$$

Therefore, motors for use with large reductions should be carefully selected. Even a small motor, if stalled, can produce enough Torque to ruin the drive, unless it is protected by a shear pin or some similar device.

Neglecting frictional losses, this sketch illustrates the manner in which Torque increases as speed decreases.



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AGMA Service Factors and Load Classifications

Type of Machine To Be Driven	Non-Motor Reducer (Service Factors) Hours Per Day 3 to 10	Non-Motor Reducer (Service Factors) Hours Per Day Over 10	Motorized Reducer (Class of Service) Hours Per Day 3 to 10	Motorized Reducer (Class of Service) Hours Per Day Over 10
AGITATORS				
Pure Liquid	1.00	1.25	I	II
Semi-Liquids, Variable Density	1.25	1.50	II	II
BLOWERS				
Centrifugal and Vane	1.00	1.25	—	—
Lobe	1.25	1.50	—	—
BREWING AND DISTILLING				
Bottling Machinery	1.00	1.25	I	II
Brew Kettles - Continuous Duty	—	1.25	—	II
Cookers - Continuous Duty	—	1.25	—	II
Mash Tubs - Continuous Duty	—	1.25	—	II
Scale Hopper - Frequent Starts	1.25	1.50	II	II
CAN FILLING MACHINES				
Can Knives	1.50	—	—	—
Car Dumpers	1.75	—	III	—
Car Pullers	1.25	—	*	—
Clarifiers	1.00	1.25	I	II
Classifiers	1.25	1.50	II	II
CLAY WORKING MACHINERY				
Brick Press & Briquette Machine	1.75	2.00	III	III
Extruders and Mixers	1.25	1.50	II	III
COMPRESSORS				
Centrifugal	1.00	1.25	—	—
Lobe - Reciprocating, Multi-Cycle	1.25	1.50	—	—
Reciprocating - Single Cycle	1.75	2.00	—	—
CONVEYORS - UNIFORMLY LOADED & FED				
Apron	1.00	1.25	II	III
Assembly Belt - Bucket or Pan	1.00	1.25	II	II
Chain - Flight	1.00	1.25	II	II
Oven - Live Roll - Screw	1.00	1.25	I	II
CONVEYORS - HEAVY DUTY NOT UNIFORMLY FED				
Apron	1.25	1.50	II	III
Assembly Belt - Bucket or Pan	1.25	1.50	II	II
Chain - Flight	1.25	1.50	II	II
Live Roll	—	—	*	*
Oven - Screw	1.25	1.50	I	II
Reciprocating - Shaker	1.75	2.00	III	III
CRANES AND HOISTS				
Main Hoists	1.00	1.25	I	II
Bridge and Trolley Drive	*	*	II	II
CRUSHER				
Ore, Stone	1.75	2.00	—	—
Sugar	1.50	1.50	—	—
ELEVATORS				
Bucket - Uniform Load	1.00	1.25	I	II
Bucket - Heavy Load	1.25	1.50	II	III
Centrifugal Discharge	1.25	1.50	I	II
Freight	1.25	1.50	II	II
Gravity Discharge	1.00	1.25	I	II
FANS				
Centrifugal - Light (Small Dia.)	1.00	1.25	—	—
Large Industrial	1.25	1.50	—	—

*Consult Manufacturer.

Type of Machine To Be Driven	Non-Motor Reducer (Service Factors) Hours Per Day 3 to 10	Non-Motor Reducer (Service Factors) Hours Per Day Over 10	Motorized Reducer (Class of Service) Hours Per Day 3 to 10	Motorized Reducer (Class of Service) Hours Per Day Over 10
FEEDERS				
Apron - Belt - Screw	1.25	1.50	—	—
Disc	1.00	1.25	—	—
Reciprocating	1.75	2.00	—	—
FOOD INDUSTRY				
Beet Slicer	1.25	1.50	II	II
Bottling, Can Filling Machines	1.00	1.25	—	—
Cereal Cooker	1.00	1.25	I	II
Dough Mixer - Meat Grinder	1.25	1.50	II	II
Generators (Not Welding)	1.00	1.25	—	—
Hammer Mills	1.75	2.00	—	—
Slicers	1.00	1.25	—	—
HOISTS				
Heavy Duty	1.75	2.00	—	—
Medium Duty and Skip Type	1.25	1.50	—	—
Laundry Tumblers	1.25	1.50	II	III
LINE SHAFTS				
Uniform Load	1.00	1.25	I	II
Heavy Load	1.25	1.50	II	II
MACHINE TOOLS				
Auxiliary Drive	1.00	1.25	I	II
Main Drive - Uniform Load	1.25	1.50	II	II
Main Drive - Heavy Duty	1.75	2.00	III	III
METAL MILLS				
Draw Bench Carriers & Main Drive	1.25	1.50	—	—
Slitters	1.25	1.50	—	—
TABLE CONVEYORS - NON REVERSING				
Group Drives	1.25	1.50	II	III
Individual Drives	1.75	2.00	III	III
Wire Drawing, Flattening or Winding	1.25	1.50	II	III
MILLS ROTARY TYPE BALL & ROD				
Spur Ring Gear and Direct Connected	—	2.00	—	—
Cement Kilns, Pebble	—	1.50	—	—
Dryers and Coolers	—	1.50	—	—
Plain and Wedge Bar	—	1.50	—	—
Tumbling Barrels	—	2.00	—	—
MIXERS				
Concrete - Continuous	1.25	1.50	II	III
Concrete - Intermittent	1.25	1.50	II	—
Constant Density	1.00	1.25	I	II
Semi-Liquid	1.25	1.50	II	II
OIL INDUSTRY				
Oil Well Pumping	—	*	—	—
Chillers, Paraffin Filter	1.25	1.50	—	—
Press Rotary Kilns	1.25	1.50	—	—
PAPER MILLS				
Agitator (Mixer)	1.25	1.50	II	II
Agitator - Pure Liquids	1.00	1.25	—	—
Barking Drums - Mechanical Barkers	1.75	2.00	—	—
Bleacher	1.00	1.25	I	II
Beater	1.25	1.50	—	—
Calendar - Heavy Duty	—	2.00	—	—

AGMA Service Factors and Load Classifications

Type of Machine To Be Driven	Non-Motor Reducer (Service Factors) Hours Per Day 3 to 10	Non-Motor Reducer (Service Factors) Hours Per Day Over 10	Motorized Reducer (Class of Service) Hours Per Day 3 to 10	Motorized Reducer (Class of Service) Hours Per Day Over 10
PAPER MILLS (Continued)				
Calendar - Anti-Friction Bearings	1.00	1.25	—	II
Cylinders	1.25	1.50	—	II
Chipper	—	2.00	—	III
Chip Feeder	1.25	1.50	—	—
Coating Rolls - Couch Rolls	1.00	1.25	—	—
Conveyors - Chips - Bark - Chemical	1.00	1.25	—	—
Conveyors - Log and Slab	—	2.00	—	—
Cutter	—	2.00	—	—
Cylinder Molds, Dryers - Anti-Friction	—	1.25	—	—
Felt Stretcher	1.25	1.50	—	II
Screens - Chip and Rotary	1.25	1.50	—	—
Thickener (AC)	1.25	1.50	—	—
Washer (AC)	1.25	1.50	—	—
Winder - Surface Type	—	1.25	—	II
PLASTICS INDUSTRY				
Intensive Internal Mixers				
Batch Type	—	1.75	—	—
Continuous Type	—	1.50	—	—
Batch Drop Mill - 2 Rolls	—	1.25	—	—
Compounding Mills	—	1.25	—	—
Calendars	—	1.50	—	—
Extruder - Variable Speed	—	1.50	—	—
Extruder - Fixed Speed	—	1.75	—	—
PULLERS				
Barge Haul	—	2.00	—	—
PUMPS				
Centrifugal	—	1.25	—	—
Proportioning	—	1.50	*	*
Reciprocating				
Single Acting, 3 or More Cycles	1.25	1.50	II	III
Double Acting, 2 or More Cycles	1.25	1.50	II	III
Rotary - Gear or Lube	1.00	1.25	I	II
RUBBER INDUSTRY				
Batch Mixers	—	1.75	—	—
Continuous Mixers	—	1.50	—	—

Type of Machine To Be Driven	Non-Motor Reducer (Service Factors) Hours Per Day 3 to 10	Non-Motor Reducer (Service Factors) Hours Per Day Over 10	Motorized Reducer (Class of Service) Hours Per Day 3 to 10	Motorized Reducer (Class of Service) Hours Per Day Over 10
RUBBER INDUSTRY (Continued)				
Continuous Mixers	—	1.50	—	—
Calendars	—	1.50	—	—
Extruders - Continuous	—	1.50	—	—
Extruders - Intermittent	—	1.75	—	—
Tire Building Machines	—	—	II	II
Tire and Tube Press Operators	—	—	I	I
SEWAGE DISPOSAL EQUIPMENT				
Bar Screens	1.00	1.25	I	II
Chemical Feeders	1.00	1.25	I	II
Collectors	1.00	1.25	I	II
Dewatering Screws	1.25	1.50	II	II
Scum Breakers	1.25	1.50	II	II
Slow or Rapid Mixers	1.25	1.50	II	II
Thickeners	1.25	1.50	II	II
Vacuum Filters	1.25	1.50	II	II
SCREENS				
Air Washing	1.00	1.25	I	II
Rotary - Stone or Gravel	1.25	1.50	II	II
Traveling Water Intake	1.00	1.25	I	II
Skip Hoists	—	—	II	—
Slab Pushers	1.25	1.50	—	—
Stokers	—	1.25	—	II
TEXTILE INDUSTRY				
Batchers or Calendars	1.25	1.50	II	II
Cards	1.25	1.50	I	II
Card Machines	1.75	2.00	III	III
Dry Cans and Dryers	1.25	1.50	II	II
Dyeing Machines	1.25	1.50	II	II
Looms	1.25	1.50	*	*
Mangles, Nappers and Pads	1.25	1.50	II	II
Soapers, Tenner Frames	1.25	1.50	II	II
Spinners, Washers, Winders	1.25	1.50	II	II
Tumbling Barrels	1.75	2.00	III	III
Windlass	1.25	1.50	II	III

*Consult Manufacturer.

This list is not all-inclusive and each application should be checked to determine if any unusual operating conditions will be encountered.

SERVICE FACTOR CHART

AGMA Class of Service	Service Factor	Operating Conditions
I	1.00	Moderate Shock - not more than 15 minutes in 2 hours. Uniform Load - not more than 10 hours per day.
II	1.25	Moderate Shock - not more than 10 hours per day. Uniform Load - more than 10 hours per day.
	1.50	Heavy Shock - not more than 15 minutes in 2 hours. Moderate Shock - more than 10 hours per day.
III	1.75 2.00	Heavy Shock - not more than 10 hours per day. Heavy Shock - more than 10 hours per day.

Application Considerations

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For most applications, select for running torque rather than starting torque. The AC motor will normally produce a 200 percent starting torque. The speed reducer is built to take at least 200% momentary overload to overcome normal starting inertia. The difference in the resulting cost can be startling.

A 20% safety factor in selection can double the life … of the speed reducer for more economy in the long run. This rule of thumb will help compensate for unexpected shock and vibration, and add substantially to wear life.

Consider "auxiliary drives" whenever possible for economy. A properly selected gear or chain drive reduction from the reducer output to the driven shaft can produce substantial savings in space and drive cost.

Avoid auxiliary drives to the input shaft … unless absolutely necessary. Auxiliary reduction from the motor to the input shaft can increase the size and cost of the drive.

Hoists

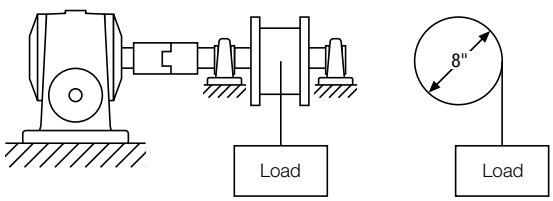
Worm gear reducers are ideal for many hoist applications. There are, however, certain precautions which should be exercised with what are thought to be self-locking characteristics of this reducer type.

A worm gear is generally said to be self-locking or irreversible when the gear cannot drive the worm – when the lead angle of the worm is less than the friction angle and hence, reverse drive efficiency is zero. This static condition can be upset by vibrations from nearby machinery or other sources. Many worm gear reducers are not self-locking, and even a particular size and ratio, which may appear to be, cannot be depended upon for this purpose. Also, a reducer which holds the load when upward movement is stopped may not when the load inertia is moving downward and the motor is stopped. For complete locking assurance, it is recommended that a fail-safe brake be used for such an application.

Finding the required torque and drum RPM…

Torque (Lb. Ins.) = (Load) x (Drum Radius)

RPM = (Velocity) ÷ .2618 x (Drum Dia.)



Belt Conveyors

Belt conveyor applications are one of those wherein the speed reducer is commonly overspecified. Proper application can, in many instances, result in substantial system economies.

To determine the torque required here, first determine the belt pull, since this is the principal force. In calculating this, the effects of sliding friction and/or angle or inclination must be considered. Table 1 shows Application Factors which may be used in determining belt pull based upon common combinations of materials and various angles.

TABLE 1. CONVEYOR APPLICATION FACTORS

Material Combinations	Angle From The Horizontal									
	0°	10°	20°	30°	40°	50°	60°	70°	80°	90°
Pivoting bucket conveyor	.025	.19	.36	.52	.66	.78	.88	.95	.99	1.00
Belt on rollers	.025	.19	.36	.52	.66	.78	.88	.95	.99	1.00
Metal on metal (finished)	.20	.37	.53	.67	.80	.89	.97	1.01	1.02	1.00
Fabric on steel	.27	.44	.60	.74	.85	.94	1.00	1.03	1.03	1.00
Fabric on wood	.32	.49	.63	.77	.88	.97	1.02	1.04	1.04	1.00
Leather on wood	.35	.52	.67	.80	.91	.99	1.04	1.06	1.05	1.00
Wood on wood	.35	.52	.67	.80	.91	.99	1.04	1.06	1.05	1.00
Plastic on steel	.35	.52	.67	.80	.91	.99	1.04	1.06	1.05	1.00
Metal on wood	.40	.57	.72	.85	.95	1.02	1.07	1.08	1.05	1.00
Rubber on wood	.45	.62	.76	.89	.99	1.05	1.09	1.09	1.06	1.00
Rubber on steel	.50	.67	.81	.93	1.03	1.09	1.12	1.11	1.07	1.00
Leather on metal	.56	.72	.87	.98	1.06	1.12	1.14	1.13	1.08	1.00

Interpolation in the table above is permissible.

The procedure involves selection of the proper Application Factor for the calculations:

Belt Pull =
(Total weight on conveyor) x (Application Factor)

Torque = (Belt Pull) x (Radius of Head Pulley)

Example:

An inclined belt conveyor is to carry cases of canned fruit. The belt is leather on a wood conveyor bed. Ten cases will be on the conveyor at a time, and each weighs 30 Lbs. The conveyor is inclined at 20° to the horizontal, and the head pulley diameter is 9".

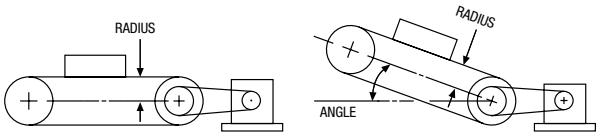
How much torque is required at the head pulley?

Select .67 as the Application Factor (Table 1)

Determine weight: 10 x 30 = 300 Lbs.

Determine belt pull: 300 x .67 = 201 Lbs.

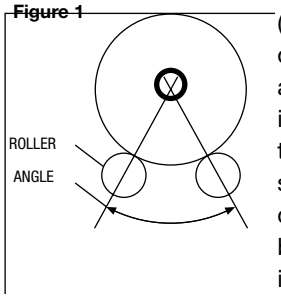
Determine Torque: 201 x 9/2 = 201 x 4.5
= 904 Lb. Ins.



Cylinders

These applications deal principally with rotation of weight about a horizontal centerline. Again, they are commonly subject to reducer overspecification. The table of Rolling Friction Factors in the section on turntables may be used, since the supporting members will be essentially the same.

Figure 1



For **Roller Supported** cylinders (Figure 1), the torque required will depend on the rolling friction factor and the angle between the rollers, in addition to the weight. Assuming that the load in the cylinder is non-solid and tends to remain essentially central (or balanced). The Table below lists angle factors to be used in the calculations.

Table 2. Angle Factors For Roller Supported Cylinders

Angle	0°	20°	40°	50°	60°	70°	80°	90°
Angle Factor	1.00	1.02	1.06	1.10	1.15	1.22	1.31	1.41

The friction force acts at the point of contact between the rollers and the cylinder, and will be:

Friction Force=(Weight)x(Friction Factor)x(Angle Factor)

Assuming the cylinder is to be driven by one of the rollers:

Torque = (Friction Force) x (Radius of Roller)

Example:

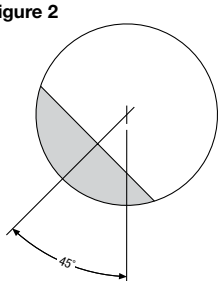
A 1200 Lb. steel cylinder is resting on two pairs of steel rollers which are 4" in diameter. The cylinder is empty. The angle between rollers is 50°. How much torque is required at the roller to turn the cylinder?

Select 1.10 as Angle Factor (Table 2), and .025 as Rolling Friction Factor (Table 3).

Determine Friction Force: 1200 x .025 x 1.10 = 33 Lbs.

Determine Torque: 33 x 2 = 66 Lb. Ins.

Figure 2



Horizontal Axis Supported cylinders (Figure 2), with unbalanced loads require a different approach. Assuming that the cylinder is not full, and that the material is rotated to a position about 45° from the vertical, the torque is equal to the Material Weight x "Effective Radius".

Effective Radius = Cylinder Diameter (D) x 0.23 (1/4 full)
0.15 (1/2 full)
0.08 (3/4 full)

Example:

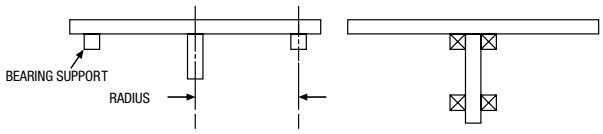
An axis supported cylinder is 3 Ft. in diameter and is half full of semi-solids mixture weighing 400 Lbs. How much torque is required (at the axis) to rotate the cylinder?

Determine the Effective Radius: .15 x (3 x 12) = 5.4 In.

Calculate Torque: 400 x 5.4 = 2160 Lb. Ins.

Q

Turntables



Here, too, turntable applications appear to lend themselves to overspecification of the speed reducer.

This type of problem involves rotation of weight in a horizontal plane, and in most cases the torque must only overcome the friction between the turntable and its supporting bearings. Assuming that the speed of rotation will be slow, the bearing loads caused by centrifugal forces (resulting from eccentric loading) may be ignored. Since in most applications, the turntable is supported by anti-friction bearings or rollers, a Table of Rolling Friction Factors is provided to be used in the following calculations.

Table 3. Rolling friction factors

Radial Ball Bearings	.001
Roller Bearings	.0015
Thrust Ball Bearings	.0034
Steel Wheels on Steel	.025
Iron Rollers on Wood	.13
Iron Rollers on Asphalt	.14

In rim supported turntables, a friction factor should be selected on the basis of the type of supporting bearings. The load on the bearings will be the sum of the weight of the turntable itself plus the load; the friction force at the bearings is the product of the total weight and the friction factor.

Force = (Total Weight x Friction Factor)

Torque = (Force) x (Radius)

Example:

A turntable 20 feet in diameter is to rotate a 4500 Lb. automobile. A ring of steel casters (riding on steel) supports the turntable, the casters are located at a radius of 8 feet. The turntable weights 1500 Lbs. How much torque is required to drive the turntable at the axis?

Select .025 as Rolling Friction Factor (Table 3).

Determine weight: 4500 + 1500 = 6000 Lbs.

Determine Friction Force: 6000 x .025 = 150 Lbs.

Determine Torque: 150 x (8 x 12) = 14,400 Lb. Ins.

Center supported turntables do not lend themselves well to calculation because the bearings are on the supporting shaft. The scale measurement of the torque will provide the most accurate value. If this is not possible, the mean radius of the bearing may be used in the above formulas with some degree of accuracy.

Application Formulas

TO OBTAIN	HAVING	FORMULA
Velocity (V) Feet Per Minute	Pitch Diameter (D) of Gear or Sprocket - Inches and Revolutions Per Minute (RPM)	$V = .2618 \times D \times RPM$
Revolutions Per Minute (RPM)	Velocity (V) Feet Per Minute and Pitch Diameter (D) of Gear or Sprocket - Inches	$RPM = \frac{V}{.2618 \times D}$
Pitch Diameter (D) of Gear or Sprocket	Velocity (V) Feet Per Minute and Revolutions Per Minute (RPM)	$D = \frac{V}{.2618 \times RPM}$
Torque (T) In. Lbs.	Force (W) Lbs. and Radius (R) Inches	$T = W \times R$
Horsepower (HP)	Force (W) Lbs. and Velocity (V) Feet Per Minute	$HP = \frac{W \times V}{33000}$
Horsepower (HP)	Torque (T) In. Lbs. and Revolutions Per Minute (RPM)	$HP = \frac{T \times RPM}{63025}$
Torque (T)	Horsepower (HP) and Revolutions Per Minute (RPM)	$T = \frac{63025 \times HP}{RPM}$
Force (W) Lbs.	Horsepower (HP) and Velocity (V) Feet Per Minute	$W = \frac{33000 \times HP}{V}$
Revolutions Per Minute (RPM)	Horsepower (HP) and Torque (T) In. Lbs.	$RPM = \frac{63025 \times HP}{T}$

Terms and Conditions

All Quotations And Sales By Boston Gear, The Contracting Party Hereto, A Division Of Altra Motion. Hereafter Called "Company" Are Made On The Following Terms And Conditions.

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1. Quotations and Their Acceptance

Unless otherwise specified, quotations on stock products are for immediate acceptance, subject to prior sales. Quotations on special products are made subject to acceptance within sixty (60) days from date thereof, but in making such quotations, the Company reserves the right to change or cancel them at any time prior to the receipt of the customers' written acceptance. All quotations for special products are based upon supplying up to plus or minus 5% of quantity ordered unless otherwise stated in the quotation. All quotations are made F.O.B. shipping point.

2. Prices

Prices are in accordance with current Company price lists, are based on quantity specified and are subject to minimum order requirements of the Company. In the event the Company consents to the cancellation or suspension of orders, it shall be entitled to charge for work done and material ordered or used up to the time of giving its written consent to such cancellation or suspension. When work is to be done on material furnished by the customer, prices are based on the quantity specified being delivered by the customer at one time within a reasonable time after acceptance of order. Quotations will be made on special products of all types or on cutting only. Prices, specifications, and terms and conditions, as well as all statements appearing in the Company's catalogs and advertisements, and made elsewhere by the Company are subject to change without notice. Changes by the customer in specifications or delivery requirements will be subject to change in price. Whenever the net price of an order amounts to less than $25.00, a minimum charge of $25.00 will be made.

3. Credit Terms

To those customer and prospective customers whose credit is satisfactory to the Company, terms are net thirty (30) days, from date of invoice, with the option of paying semi-monthly. The Company may at any time when, in its opinion, the financial condition of the customer or prospective customer warrants it, either alter or suspend credit, or discontinue deliveries, and render a charge covering the value of any partially finished special products which are then being manufactured for the customer. In those instances where credit is not established, and in cases where satisfactory references are not given, the terms are cash with order. For special products in those instances where credit is not established to the satisfaction of the Company, a deposit of at least 50% of total value of the order is required. Remittances should be made by check or money order, payable to the Boston Gear, P.O. Box 521 South Beloit, IL 61080, U.S.A. Delays in transportation shall not exceed the terms of payment.

4. Material Furnished by The Customer

Unless otherwise specified, quotations are based on material furnished by the customer being of ordinary hardness, normal allowance for finish, uniform specification, and machine work being of ordinary commercial accuracy. If material furnished by the customer involves the Company in expense not contemplated by the contract, the customer will be charged for all such additional expense. If serious defects are found in the material furnished by the customer, the customer will be charged for the actual work done. The Company assumes no responsibility for, and will not be liable for loss of or damage to samples, blueprints, diagrams, and other material of any nature submitted or furnished by the customer or prospective customer, provided the Company has exercised reasonable care in the handling of the same. The Company does not assume transportation and insurance costs on any of the foregoing items. In all cases where the customer or prospective customer makes no statement in writing, concerning the disposition of any of the foregoing material when submitted, the Company reserves the right to dispose of such material according to its best judgement.

5. Dimensions

When dimensions of rims, bores, and hubs are not clearly specified, quotations are based on ordinary dimensions. Before the customer's blanks are accepted by the Company for cutting, the diameter, holes, rims, and ends of holes must be finished; for bevel gears, hubs, must be of uniform length. There should also be an allowance of extra blanks to cover possible spoilage. Unless otherwise specified, dimensions are in inches.

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6. Samples

In no case are samples furnished free. If agreed to by the Company, a few products in advance of a regular quantity order will be furnished but only at an agreed upon price over the regular quantity price.

7. Taxes

If any tax is at any time levied or imposed by the federal or any state or local government, or any other taxing authority, upon the products covered hereby, or in respect of the production, processing, manufacture, storage, sale, use, or consumption thereof, or, in the case of goods delivered at the Company's expense, upon the transportation thereof, including freight charges thereon, the amount of such tax shall be added to the purchase price above specified and shall be borne by the customer. The Company will accept a valid exemption certificate from the customer if applicable; however, if any exemption certificate previously accepted is not recognized by the taxing authority involved and the Company is required to pay the tax covered by such exemption certificate, the customer shall be required to promptly reimburse the Company for the taxes so paid.

8. Shipments

All shipments are made F.O.B. shipping point (subject to freight allowance under conditions stated in separate price schedules). When ordering, the customer's desired method of shipment must be clearly stated. Where instructions for shipping do not appear on the order, shipment will be made according to the Company's best judgment. Full risk of loss (including transportation delays and losses) shall pass the customer upon delivery of the products to F.O.B. point. Unless otherwise instructed, all Parcel Post shipments are insured at the customers' expense. Parcel Post shipments without insurance are at the customer's risk. Deliveries by Messenger Service to a terminal are made at the customer's risk and expense. Partial shipments shall be permitted and the Company may invoice each shipment separately.

9. Refusal of Shipment

In case of the refusal or inability of the customer to accept any shipment in accordance with the terms of the order, the customer shall be liable for freight, express, storage, extra cost of handling and all other expenses incurred by the Company as a result of such refusal or inability.

10. Delay or Nonperformance

The Company shall not be liable for any delay or loss of any nature or failure in performance due to or caused by fire, flood, strike, or other differences with workmen, accidents, labor or material or transportation shortages, war (declared or undeclared), insurrection, riot, or by any governmental orders or regulations, legal interferences or prohibitions, defaults on the part of suppliers or other causes beyond the Company's reasonable control.

11. Claims and Rejected Material

Any products which have been altered or damaged are not returnable except with the Company's written consent. To reject products on inspection as defective, customer must notify the Company in writing within ten (10) days from receipt of the products. Before allowing or rejecting claim, the Company shall then have the option of reinspection at the customer's plant or its own. Defects that do not impair service shall not be a cause for rejection. The Company shall have the right to replace within a reasonable time any product or products which in its opinion do not conform to the order. No claim will be allowed for any products damaged by the customer or damaged in transit. Expenses incurred in connection with claims for which the Company is not liable, will be charged to the customer. The Company will not be responsible for any work done to correct errors unless such work is authorized by the written consent of the Company. The Company assumes no liability for any claim for infringement of any foreign or domestic patent.

12. Limited Warranty

The Company warrants that products manufactured or sold by it shall be free from defects in material and workmanship. Any products which shall within two (2) years of delivery, be proved to the Company's satisfaction to have been defective at the time of delivery in these respects will be replaced or repaired by the Company at its option. Freight is the responsibility of the customer. The Company's liability under this limited warranty is limited to such replacement or repair and it shall not be held liable in any form of action for direct or consequential damages to property or person. THE FOREGOING LIMITED WARRANTY IS EXPRESSLY MADE IN LIEU OF ALL OTHER WARRANTIES WHATSOEVER, EXPRESS, IMPLIED AND STATUTORY AND INCLUDING WITHOUT LIMITATION THE IMPLIED WARRANTIES OF MERCHANTABILITY AND FITNESS.

No employee, agent, distributor, or other person is authorized to give additional warranties on behalf of Boston Gear, nor to assume for Boston Gear any other liability in connection with any of its products, except an officer of Boston Gear by a signed writing

Terms and Conditions

All Quotations And Sales By Boston Gear, The Contracting Party Hereto, A Division Of Altra Motion. Hereafter Called "Company" Are Made On The Following Terms And Conditions.

13. Waiver of Breach

No waiver by the Company of any breach of these provisions shall constitute a waiver of any other breach.

14. Consequential Damages

The Company shall not be liable to the customer or others claiming through the customer for special or consequential charges for any reason whatsoever.

15. Laws

To the best of the Company's knowledge and belief it is in compliance with all local, state and federal laws. All orders are subject to the condition that the Company's obligation under such local, state and federal laws and Executive Orders, Rules and Regulations issued thereunder, whether now in force or hereafter made effective, shall be no greater as a result of this agreement and no greater than required by such laws and the Company expressly disclaims assumptions of any of the customer's obligations under such laws.

16. General

Any terms and conditions of a customer's order which are inconsistent with or additional to the terms and conditions hereof shall not be binding on the Company and shall not be considered applicable to any sale or shipment of the Company's products. All such terms and conditions are hereby expressly rejected. No waiver, alteration or modification of any of the Company's terms and conditions shall be binding on the Company unless made in writing and agreed to by a duly authorized official of the Company.

17. Printers, Stenographic, and Clerical Errors

The Company is not responsible for printers' errors made in any of its publications and other forms of printed matter, or for any stenographic and clerical errors. All such errors are subject to correction.

18. Reducer Express

- Quantities of reducers covered as part of this program are a maximum of:
  6 pieces for any 710-726 or 221-231 and 832-843
  2 pieces for any 730-760 or 239-247 and 852-873
- Bost-Kleen, Stainless Bost-Kleen and modified reducers are not included as part of this program.
- Boston Gear will utilize any major courier to handle air shipments.
- Consult Boston Gear for details.

19. Guaranteed Same Day Shipment

- Products must be available from stock.
- Does not apply to WOG or scheduled release shipments.
- Same day shipment available Monday through Friday excluding U.S. holidays. For emergency service, please call 704-688-7350.
- In the event your freight carrier is unable to meet your requirements, we reserve the right to substitute a carrier of equivalent quality.
- If a shipment is missed and Boston Gear pays the freight, we'll pay for the freight charges as they were originally specified on the order.
- Brokerage and export fees still apply to shipments outside the U.S.
- Video Terminal Orders entered up to 8 p.m. Eastern Time will be shipped the same day.